CONCERNING THE GENERAL EQUATIONS

OF TURBULENCE

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General equations of turbulent flow are derived for a fluid with large gradients of drift velocities.

1. In order to describe the turbulent flow of a liquid or a gas, one usually begins with the Navier – Stokes equations. We note, however, that these equations have a definite range of applicability and cannot always be used as a basis for deriving the equations of turbulence. This is the case, for example, when the gradients of drift velocities of molecules in a liquid or a gaseous stream vary appreciably over the characteristic hydrodynamic dimension of the container. It then becomes necessary to modify the equations of hydrodynamics. Such a generalization was made by Predvoditelev [1]. The equations of hydrodynamics which he has derived describe such complex hydrodynamic situations as may arise, for example, in a stream of rarefied gas, in a stream near surfaces, and in stellar systems.

The system of hydrodynamic equations which has been derived in [1] for a viscous incompressible fluid is

$$\rho \left[\frac{\partial v_{\alpha}}{\partial t} + (1 - \beta) v_{\gamma} \frac{\partial v_{\alpha}}{\partial x^{\gamma}} \right] = -\frac{\partial p}{\partial x^{\alpha}} + \mu \frac{\partial^2 v_{\alpha}}{\partial x^{\gamma} \partial x^{\gamma}} , \qquad (1)$$

and the continuity equation does not change:

$$\frac{\partial v_{\gamma}}{\partial x^{\gamma}} = 0. \tag{2}$$

Parameter β in (1), which characterizes the deviation from ideal continuity, can be expressed in terms of the Knudsen number and the Mach number

$$|\beta| = \frac{3}{2} \operatorname{Kn} \mathcal{M}. \tag{3}$$

Starting from the general equations of hydrodynamics (1) and (2), we will derive the equations of turbulent fluid flow according to Reynolds [2, 3].

2. We will consider [2, 3] the component u of instantaneous velocity to consist of the average velocity component \overline{u} and the turbulent pulsating velocity component u', so that

 $u = u + u', \quad v = v + v', \quad w = w + w'.$ (4)

Then, averaging in accordance with Reynolds' rules, we obtain a system of Reynolds equations for the general equations of hydrodynamics:

$$\rho \left[\frac{\partial \overline{u}}{\partial t} + (1 - \beta) \left(\overline{u} \frac{\partial \overline{u}}{\partial x} + \overline{v} \frac{\partial \overline{u}}{\partial y} + \overline{w} \frac{\partial \overline{u}}{\partial z} \right) \right] = -\frac{\partial p}{\partial x} + \mu \Delta \overline{u} + (1 - \beta) \left[\frac{\partial}{\partial x} \left(-\overline{\rho u'^2} \right) + \frac{\partial}{\partial y} \left(-\overline{\rho u'v'} \right) + \frac{\partial}{\partial z} \left(-\overline{\rho u'w'} \right) \right],$$

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$$\rho \left[\frac{\partial \overline{v}}{\partial t} + (1 - \beta) \left(\overline{u} \frac{\partial \overline{v}}{\partial x} + \overline{v} \frac{\partial \overline{v}}{\partial y} + \overline{w} \frac{\partial \overline{v}}{\partial z} \right) \right] = -\frac{\partial p}{\partial y} + \mu \Delta \overline{v} + (1 - \beta) \left[\frac{\partial}{\partial x} (-\rho \overline{u'v'}) + \frac{\partial}{\partial y} (-\rho \overline{v'^2}) + \frac{\partial}{\partial z} (-\rho \overline{w'v'}) \right],$$

$$\rho \left[\frac{\partial \overline{w}}{\partial t} + (1 - \beta) \left(\overline{u} \frac{\partial \overline{w}}{\partial x} + \overline{v} \frac{\partial \overline{w}}{\partial y} + \overline{w} \frac{\partial \overline{w}}{\partial z} \right) \right]$$

$$= -\frac{\partial p}{\partial z} + \mu \Delta \overline{w} + (1 - \beta) \left[\frac{\partial (-\rho \overline{u'w'})}{\partial x} + \frac{\partial (-\rho \overline{v'w'})}{\partial y} + \frac{\partial (-\rho \overline{w'^2})}{\partial z} \right],$$

$$\frac{\partial \overline{u}}{\partial x} + \frac{\partial \overline{v}}{\partial y} + \frac{\partial \overline{w}}{\partial z} = 0.$$
(6)

Letting parameter β be equal to zero in (5), we obtain the classical Reynolds system of turbulent-flow equations based on the Navier-Stokes equations:

$$\rho\left(\frac{\partial \overline{u}}{\partial t} + \overline{u} \frac{\partial \overline{u}}{\partial x} + \overline{v} \frac{\partial \overline{u}}{\partial y} + \overline{w} \frac{\partial \overline{u}}{\partial z}\right) = -\frac{\partial p}{\partial x} + \mu\Delta\overline{u} + \left[\frac{\partial}{\partial x}\left(-\rho\overline{u'^{2}}\right) + \frac{\partial}{\partial y}\left(-\rho\overline{u'v'}\right) + \frac{\partial}{\partial z}\left(-\rho\overline{u'w'}\right)\right],$$

$$\rho\left(\frac{\partial \overline{v}}{\partial t} + \overline{u} \frac{\partial \overline{v}}{\partial x} + \overline{v} \frac{\partial \overline{v}}{\partial y} + \overline{w} \frac{\partial \overline{v}}{\partial z}\right) = -\frac{\partial p}{\partial y} + \mu\Delta\overline{v} + \left[\frac{\partial}{\partial x}\left(-\rho\overline{u'v'}\right) + \frac{\partial}{\partial y}\left(-\rho\overline{v'^{2}}\right) + \frac{\partial}{\partial z}\left(-\rho\overline{w'v'}\right)\right],$$

$$\rho\left(\frac{\partial \overline{w}}{\partial t} + \overline{u} \frac{\partial \overline{w}}{\partial x} + \overline{v} \frac{\partial \overline{w}}{\partial y} + \overline{w} \frac{\partial \overline{w}}{\partial z}\right) = -\frac{\partial p}{\partial z} + \mu\Delta\overline{w} + \left[\frac{\partial}{\partial x}\left(-\rho\overline{u'w'}\right) + \frac{\partial}{\partial y}\left(-\rho\overline{v'w'}\right) + \frac{\partial}{\partial z}\left(-\rho\overline{w'^{2}}\right)\right].$$
(7)

A comparison between systems (5) and (7) shows that the difference between the right-hand sides of the turbulent-flow equations derived here and those the classical equations consists of a change in the Reynolds tensor of turbulent stresses. The additional components of this tensor are proportional to the parameter β . In the applications mentioned here these additional components may be quite significant. The problem of closing system (5), (6) is complex and calls for a separate analysis.

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